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# Gears 1

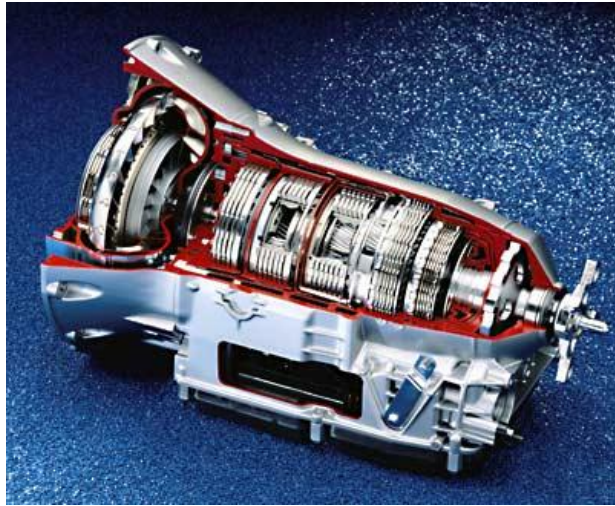
## Introduction to Gears

**Dr Hengan Ou**

Coates B80a

[h.ou@nottingham.ac.uk](mailto:h.ou@nottingham.ac.uk)

# Why gears?



MB CLK, automatic transmission

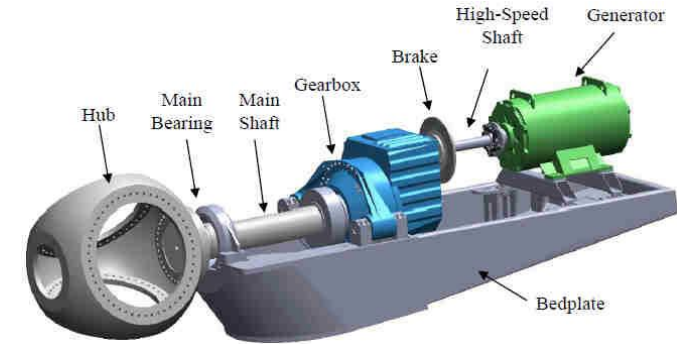
An earlier version of Nissan Leaf Gearbox



Automotive differential unit



Geared turbofan for new type of aeroengine



Modular drive train configuration  
<http://www.nrel.gov/wind/>



Vestas V90-3MW wind turbine  
<http://www.vestas.com/en>

# Gears related Lecture sessions

## Gears 1

- **Introduction to gears**
  - Functions & types
  - Gear fundamentals, terminologies & conjugate action
  - Involute profile, fundamental equations, tooth system

## Gears 2

- **Gear trains and their applications**
  - Simple train and compound train
  - Planetary train
  - Differential unit
  - Applications

## Gears 3

- **Gear stress analysis & design**
  - Common forms of gear failure
  - Gear force analysis
  - AGMA based gear design calculations

# Learning objectives

- To be familiar with **different gear types** for different applications
- To understand **gear fundamentals** and to be familiar with **gear terminology**
- To be able to determine **gear ratios** and to design different **gear trains**
- To be able to determine **AGMA bending and contact stresses** of spur gears
- To understand the working mechanisms of gear systems in applications

To understand **gear fundamentals** and to be able to **design and evaluate a gear system for specific applications.**

**Notes:** topics on **gear manufacturing** will not be covered.

# Outline of Gears 1

- **Introduction to gears**

**Part 1:**

- Functions & types of gears
- Gear terminologies

**Part 2:**

- Involute tooth profile
- Concept of conjugate action
- Gear ratio equation and tooth system

**Part 3:**

- A few worked examples



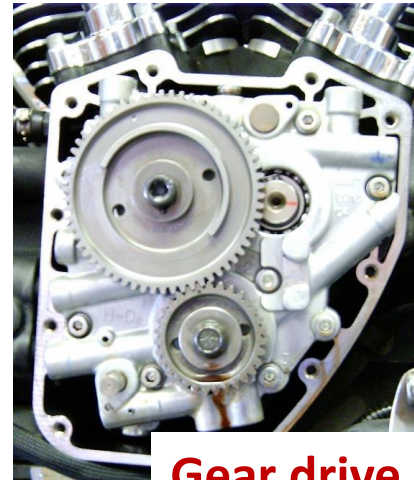
# Functions and types

## Functions:

- Transmission of power or motion between two shafts
- Most rugged and durable means of power transmission
- High transmission efficiency

## Classification of gear types:

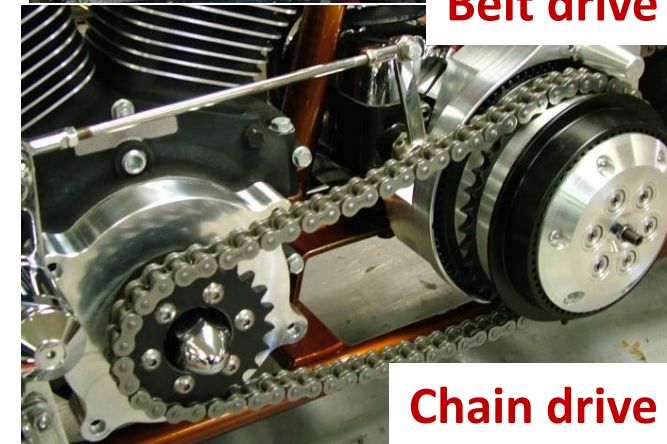
- Gears between **parallel shafts**, e.g. spur & helical gears
- Gears between **intersecting shafts**, e.g. bevel gears
- Gears between **nonparallel & non-intersecting shafts**, e.g. worm gears, hypoid gears



**Gear drive**



**Belt drive**

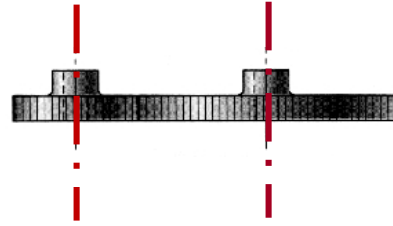


**Chain drive**

# Gear types

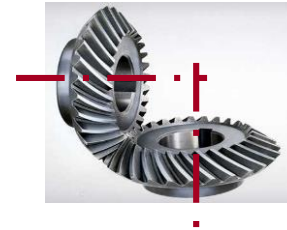
## 1. Parallel axis gears

- Spur gears
- Helical gears
- Internal gears



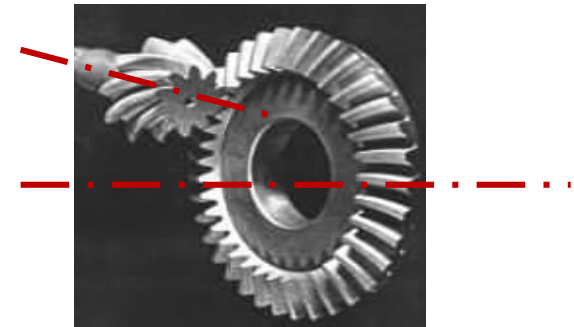
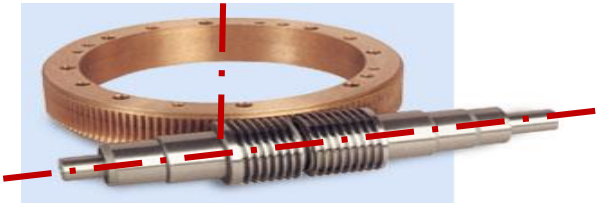
## 2. Non-parallel axis, coplanar gears (intersecting axes)

- Bevel gears
- Face gears
- Conical involute gears



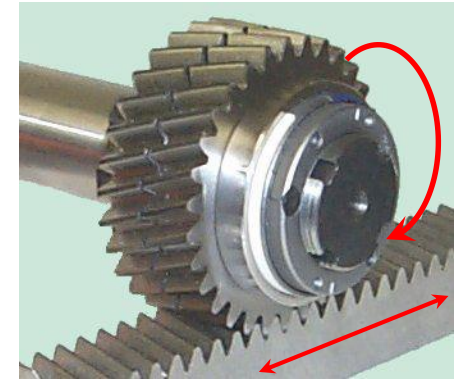
## 3. Non-parallel axis, non-coplanar gears (non-intersecting axes)

- Worm gears
- Hypoid gears



# Spur, helical and herringbone gears (between parallel shafts)

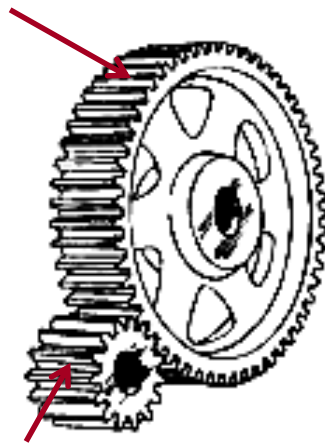
- **Spur:** tooth members parallel to shaft axes
- **Helical:** tooth members cut with an inclined angle to shaft axes, allowing smoother and quieter meshing but with an axial load generated



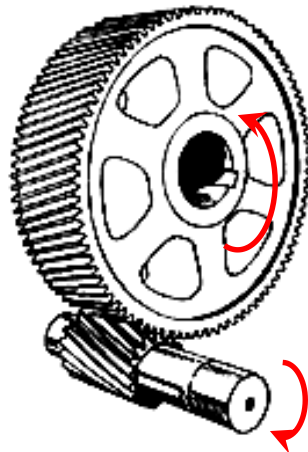
Rack and pinion

**Gear or wheel**

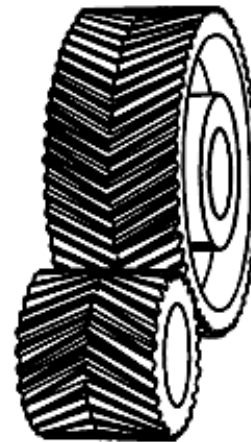
**Pinion**



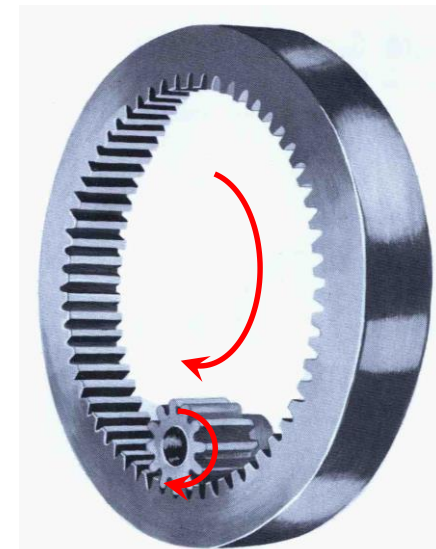
Spur gears



Helical gears



Herringbone gears  
(double helical gears)



Internal gear meshing

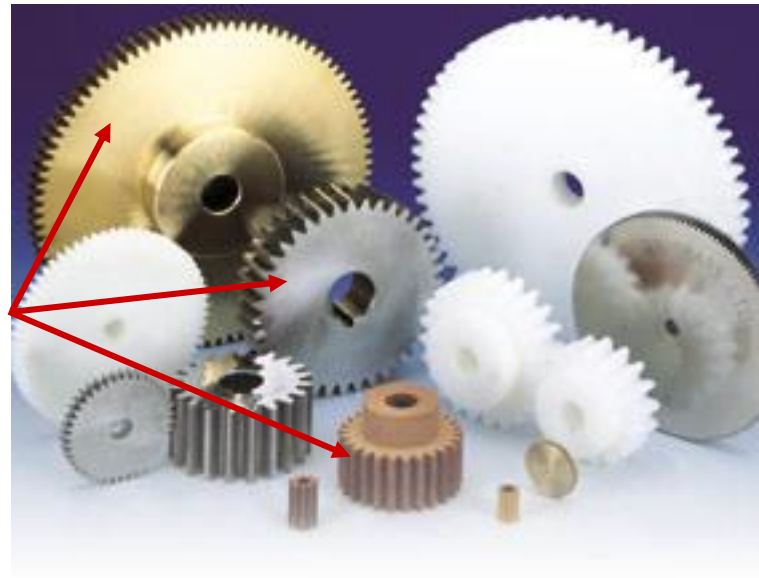


# Spur gears (between parallel shafts)

- Most common type of gears
- Cheaper to manufacture
- Generate more noise
- Smaller number of teeth in contact at any given time
- Sensitive to alignment



**Different materials**



# Helical gears (between parallel shafts)

- Teeth cut with an inclined angle to the axis of rotation
- Contact between two teeth more progressive and longer is less noisy and carries higher loads
- Can be mounted at right angles **if cut** at appropriate angle
- **Herringbone (double helical) gears** cancel out the thrust in helical gears & allow smoother power transmission at high speed



Herringbone gears

# Bevel gears (between intersecting shafts)

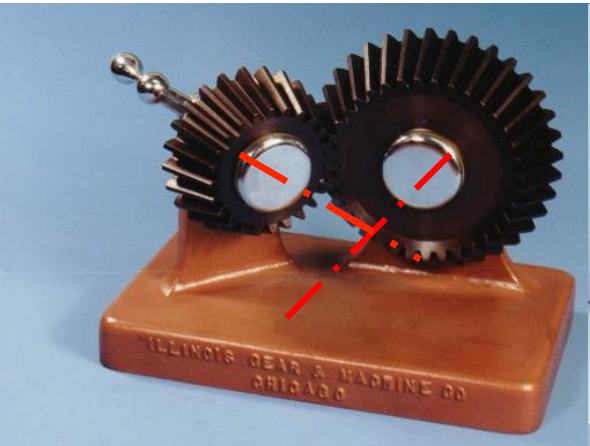
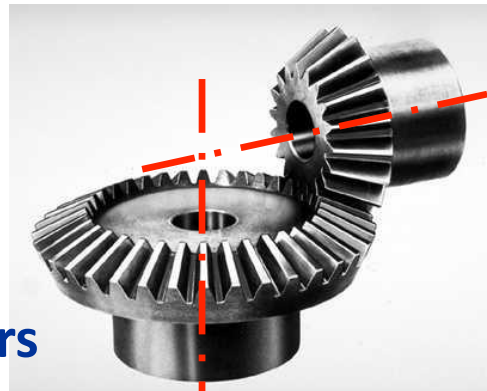
- **Straight:** ordinary tooth profile but formed on conical surfaces, different intersecting angles may be possible
- **Spiral:** similar to helical gears with more gradual meshing



Spiral bevel gears



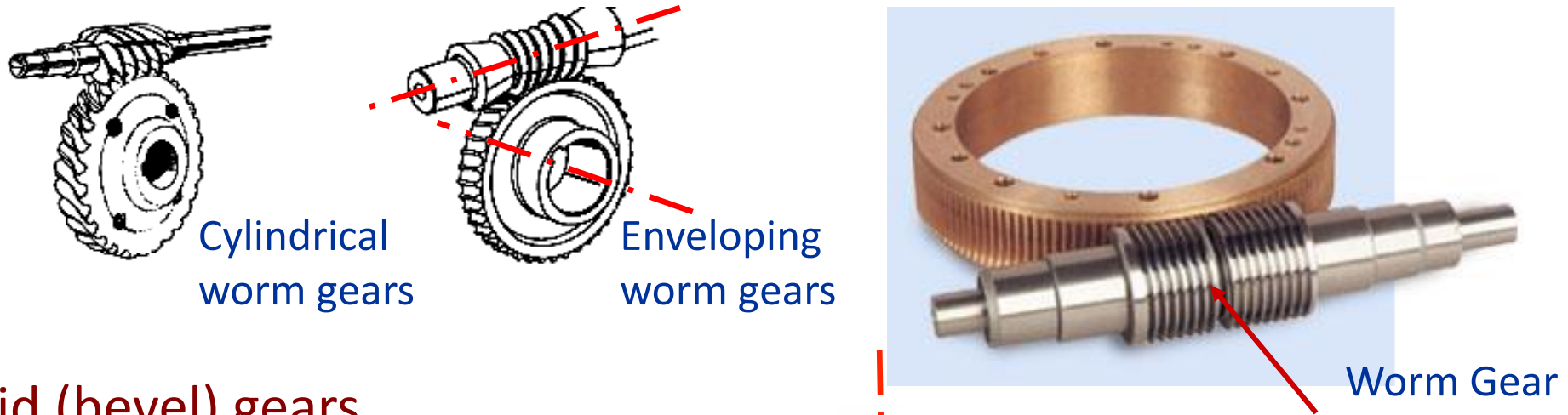
Straight bevel gears





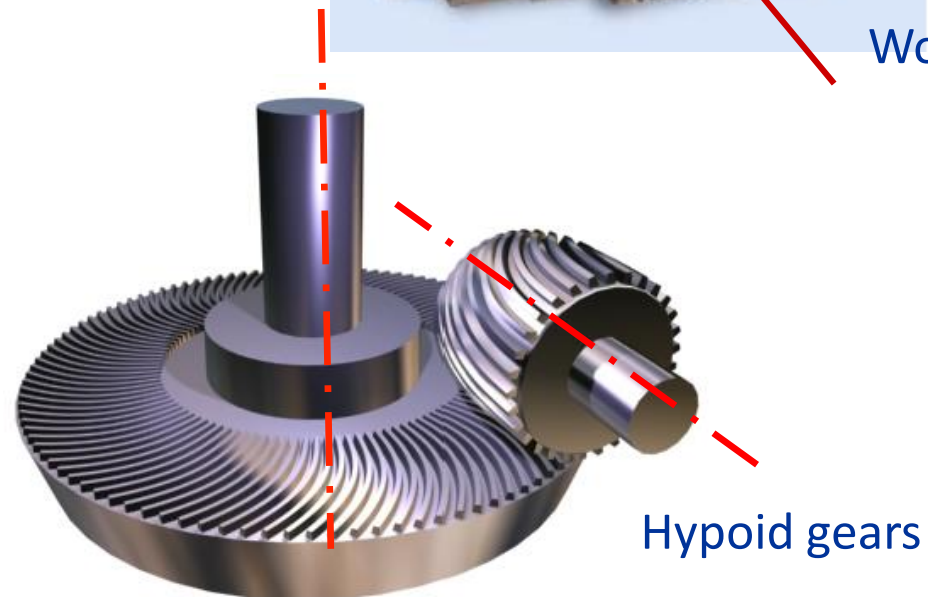
# Worm gears (between non-parallel & non-interacting shafts)

- Essentially a screw meshing with a special helical gear to provide high reduction ratio but with lower efficiency due to high sliding velocity,



## Hypoid (bevel) gears

- Can be used between non-intersecting shafts
- Typically found in automotive differentials

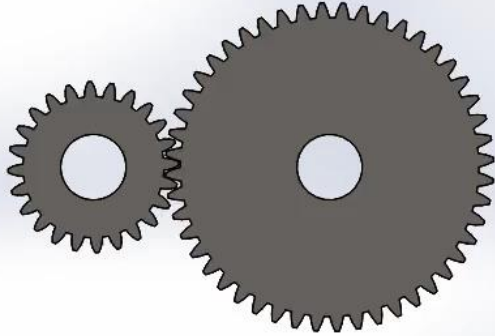


# SW based animation of different gear types in operation

Gears 1 Part 1

Spur gears

$N_p=24, N_g=48$



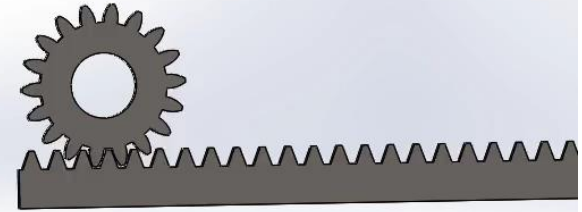
Internal spur gears

$N_p=18, N_g=72$



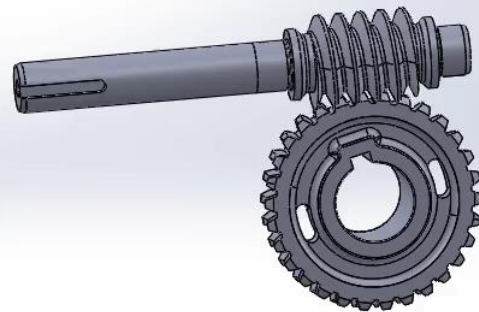
Rack & pinion

$N_p=20$



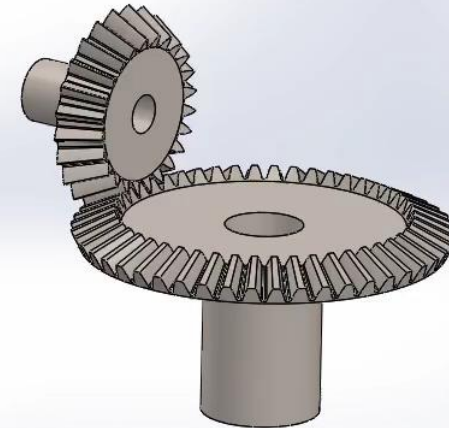
Worm & gear

$N_g=30$



Bevel gears

$N_p=24, N_g=48$



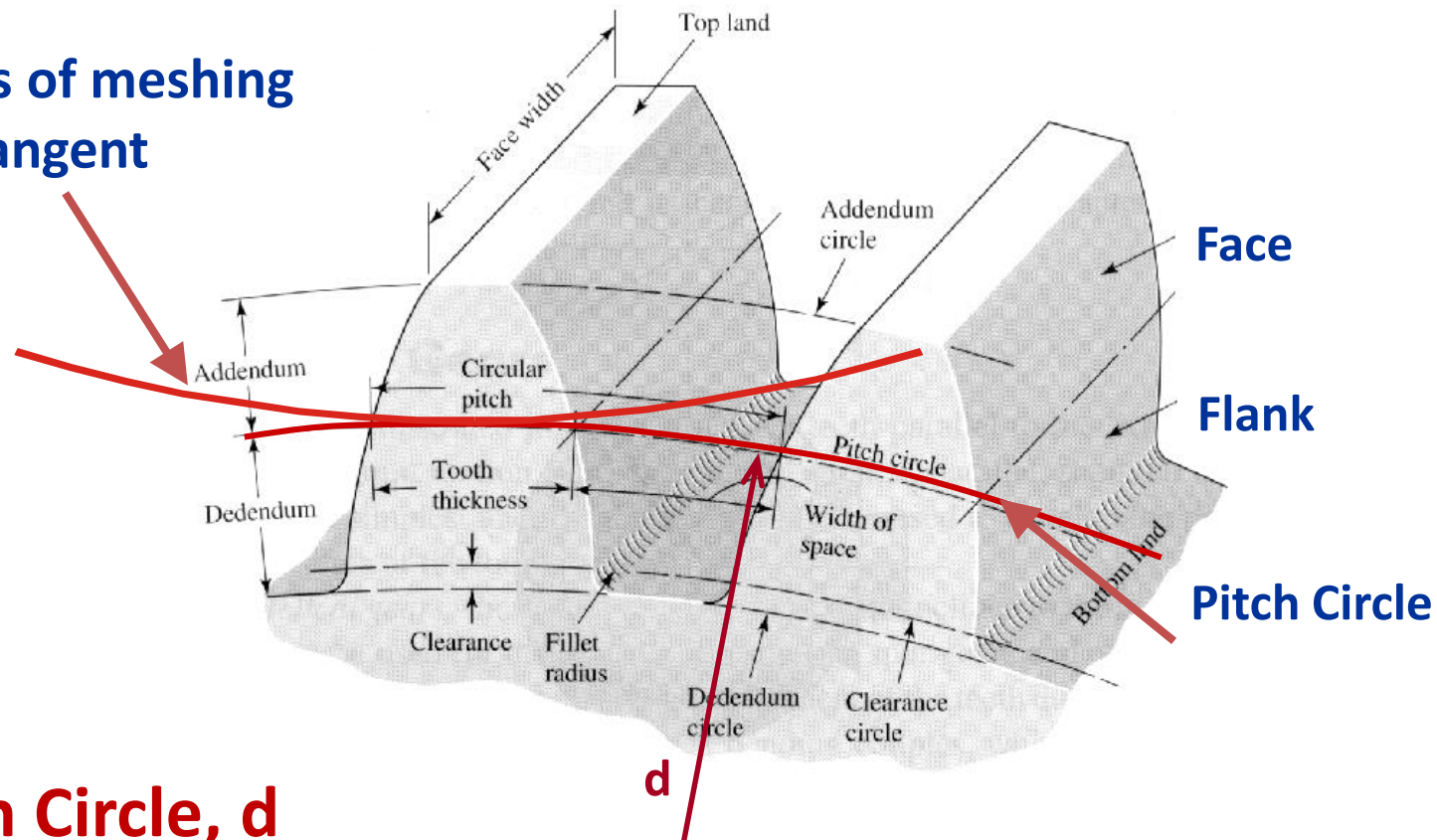
**Note:** SW assembly models of different gear types are available on Moodle



# Gear terminologies

(for design evaluation, e.g. gear ratios & motion, load carrying capability & performance)

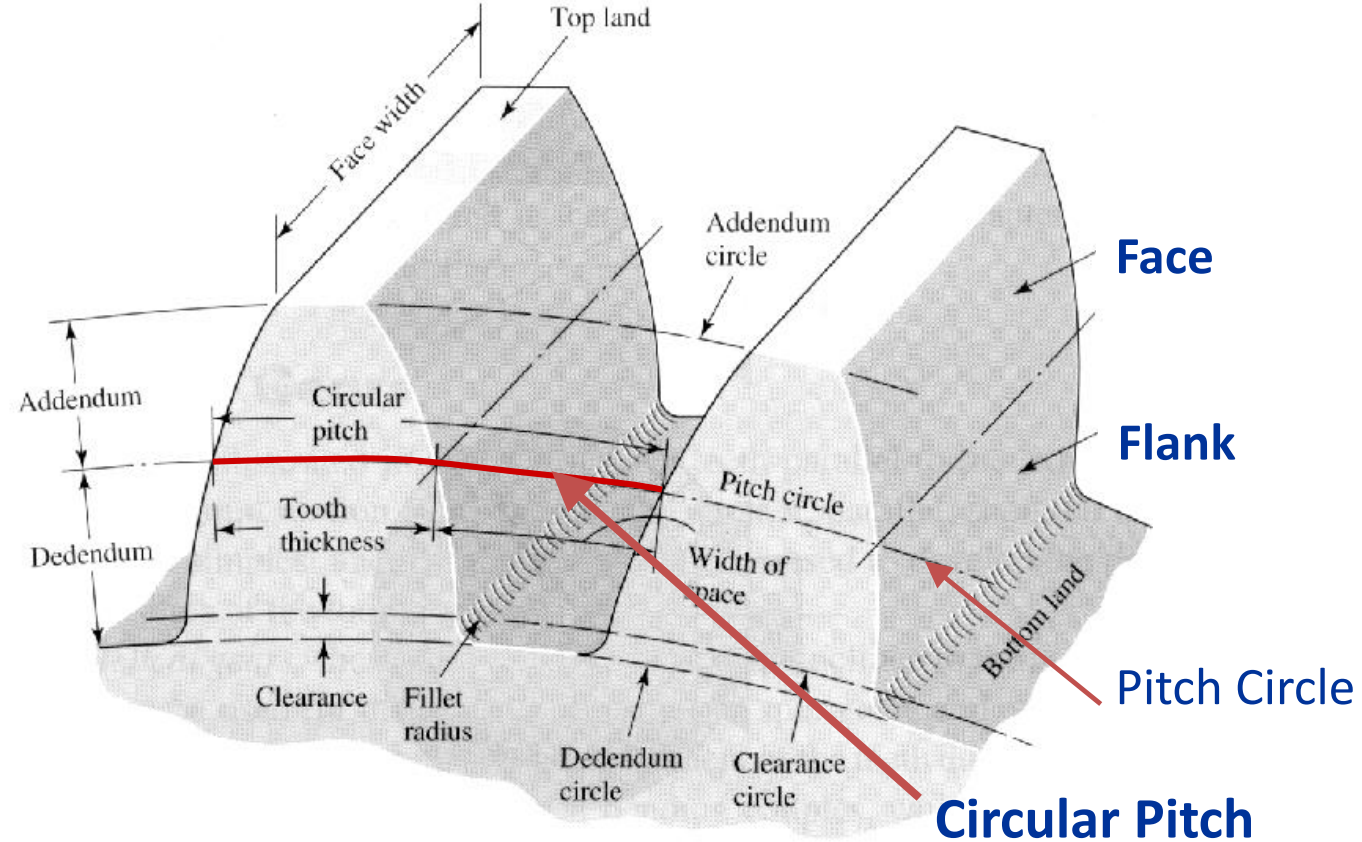
Pitch circles of meshing gears are tangent



**Pitch Circle,  $d$**

- Theoretical circle upon which all calculations are based
- Has a pitch diameter =  $d$

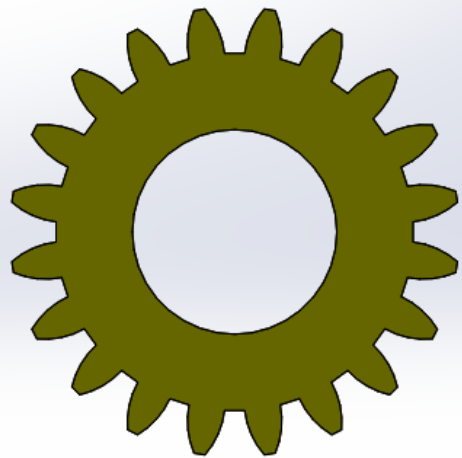
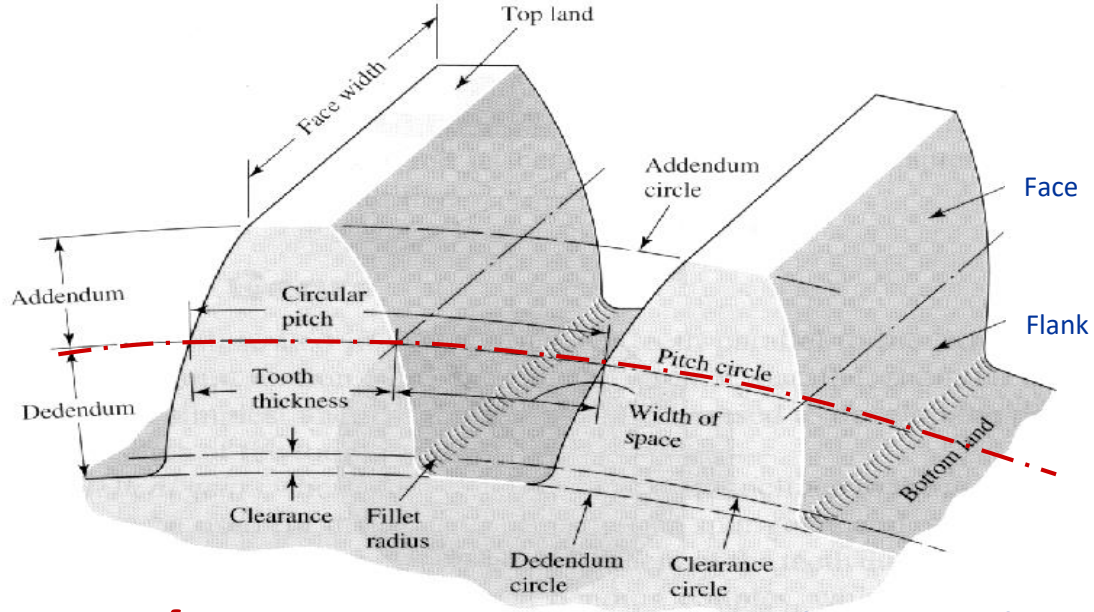
# Gear terminology



## Circular Pitch, $p = \pi d / N$

- The distance between a point on a tooth and the same point on the adjacent tooth
- Measured on the pitch circle

# Gear terminology



**d = 80 mm**  
**N = 20**



**d = 80 mm**  
**N = 40**

**m = d/N = 4 mm**

**m = d/N = 2 mm**

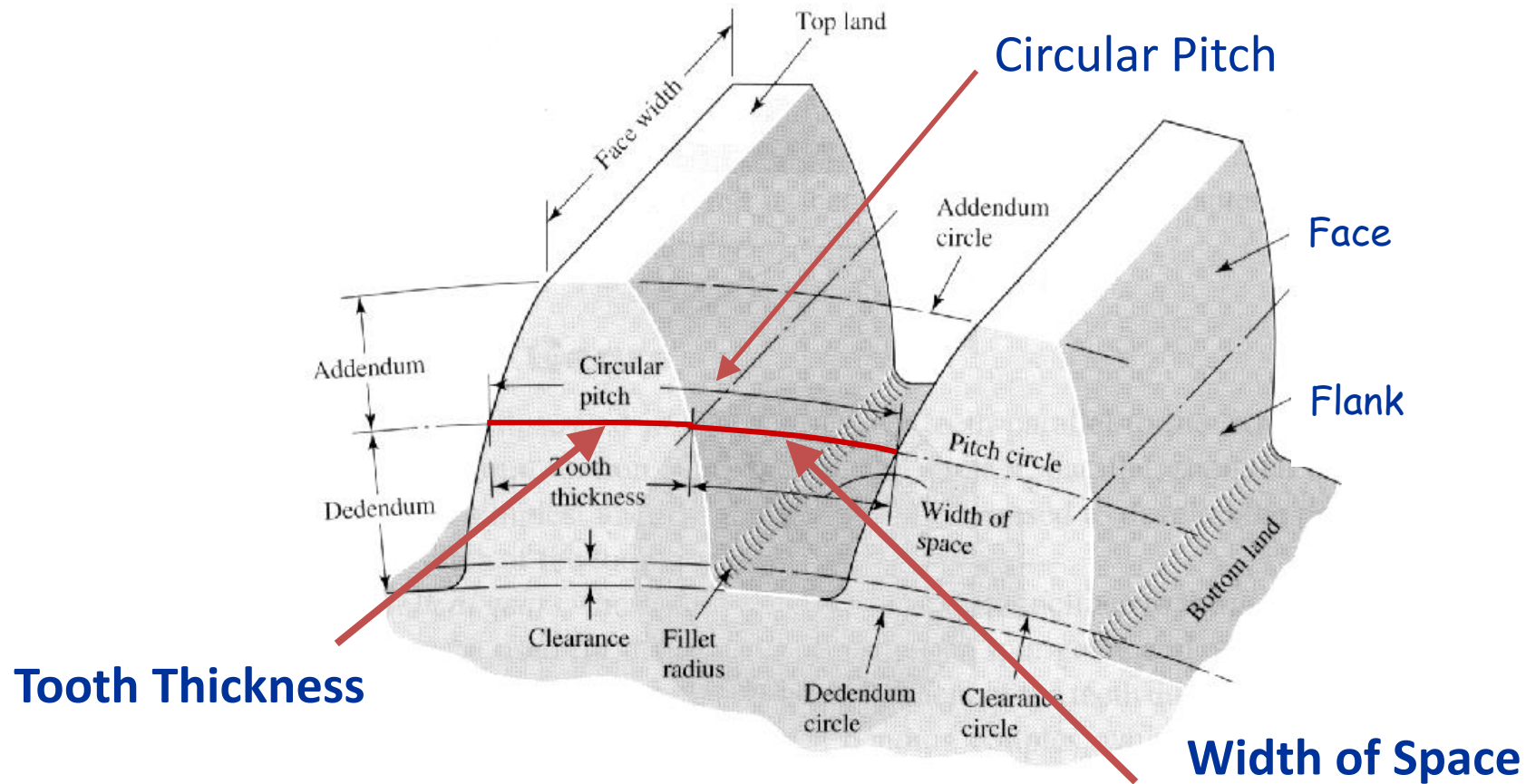
**Module,  $m=d/N$ , measured in mm (SI unit)**

- The ratio of the pitch diameter **d** to the number of teeth **N**
- Also circular pitch,  **$p = \pi d/N = \pi m$**
- **m** represents is the **size of gear tooth** & is **standardised**

**Diametral pitch,  $P=N/d$ , measured in 1/in (imperial unit)**



# Gear terminology



**Tooth Thickness,  $t = p/2 = \pi m/2$**

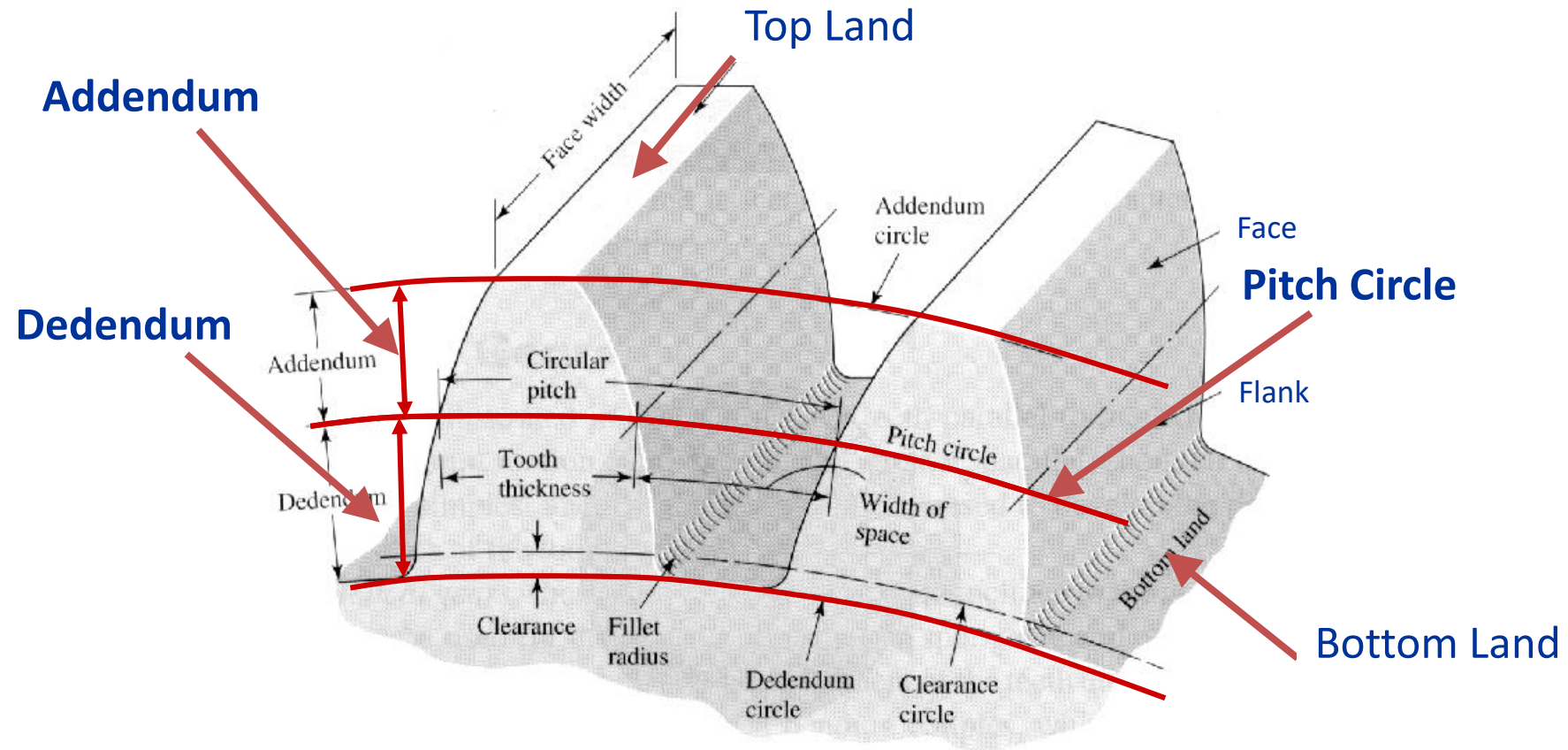
- Thickness of the gear tooth
- Measured on the pitch circle

**Width of Space  $= p/2 = \pi m/2$**

- Distance between gear teeth
- Measured on the pitch circle



# Gear terminology



**Addendum,  $a = m$**

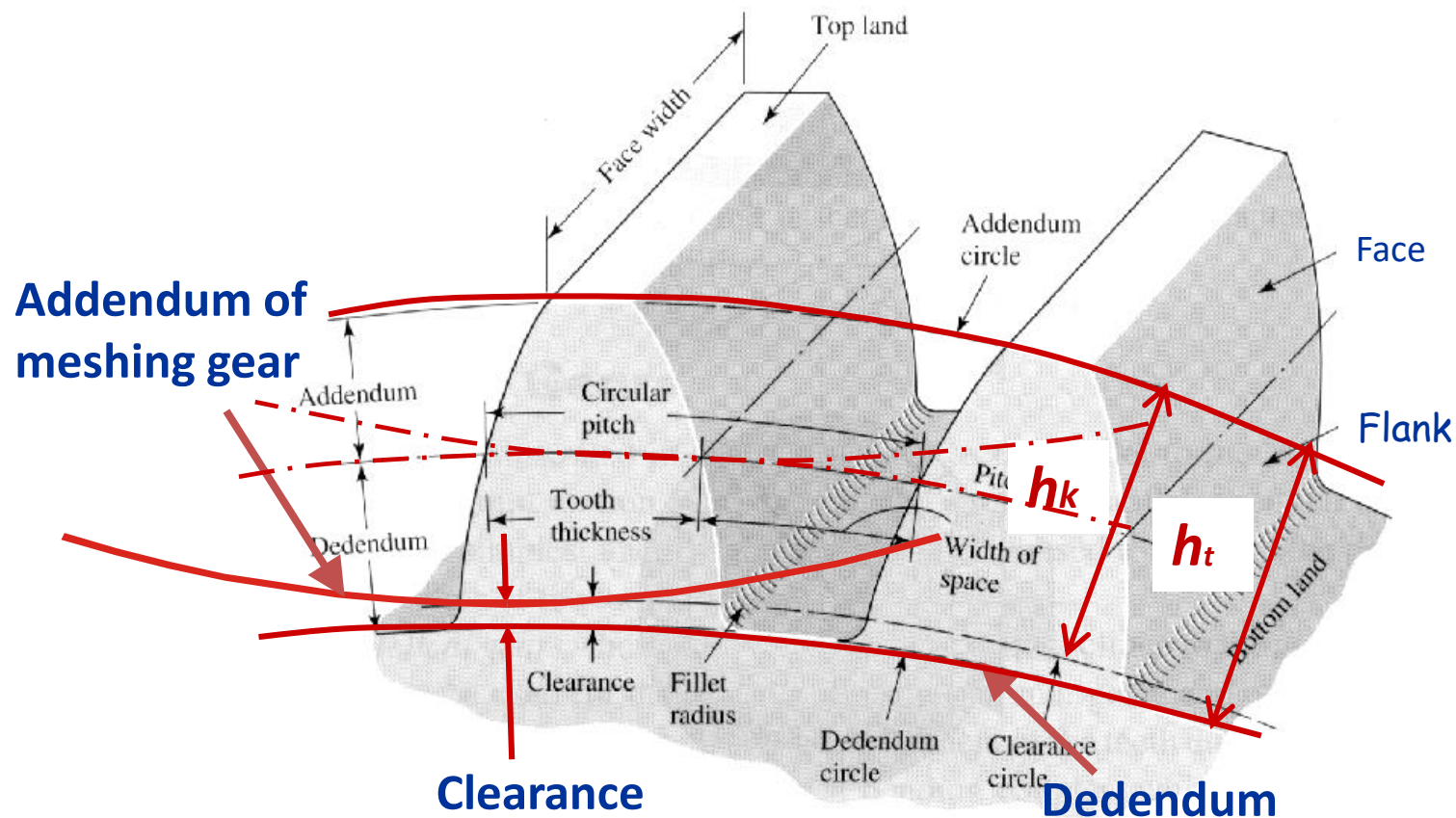
- Radial distance between the pitch circle and the top land
- Defined by Addendum Circle

**Dedendum,  $b = 1.25m$**

- Radial distance between the pitch circle and the bottom land
- Defined by Dedendum Circle



# Gear terminology



Clearance,  $c=0.25m (=b-a)$

- Radial distance between the dedendum circle and the addendum circle of the meshing gear

Whole depth,  $h_t=a+b=2.25m$

Working depth,  $h_k=a+b-c=2.0m$

# Quiz 1: True or False to each of the following statements

## Gears 1

- A. The directions of rotation of two external helical gears are opposite to each other.
- B. Bevel gears can be used for power transmission between two non-parallel and non-intersecting shafts
- C. Module ( $m$ ) is a fundamental parameter to specify tooth size defined by the ratio of pitch diameter( $d$ ) and number of teeth ( $N$ ), i.e.  $m=d/N$
- D. The Tooth thickness and Width of space are equal only on the pitch circle.
- E. The Working depth ( $h_k$ ) of a gear is the sum of Addendum ( $a$ ) and Dedendum ( $b$ )

# Quiz 1: True or False to each of the following statements

## Gears 1

- A. The directions of rotation of two external helical gears are opposite to each other. (true)
- B. Bevel gears can be used for power transmission between two non-parallel and non-intersecting shafts (false)
- C. Module ( $m$ ) is a fundamental parameter to specify tooth size defined by the ratio of pitch diameter( $d$ ) and number of teeth ( $N$ ), i.e.  $m=d/N$  (true)
- D. Tooth thickness and Width of space are equal only on pitch circle. (true)
- E. The Working depth ( $h_k$ ) of a gear is the sum of Addendum ( $a$ ) and Dedendum ( $b$ ) (false)



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# Gears 1

## End of Part 1



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# Gears 1

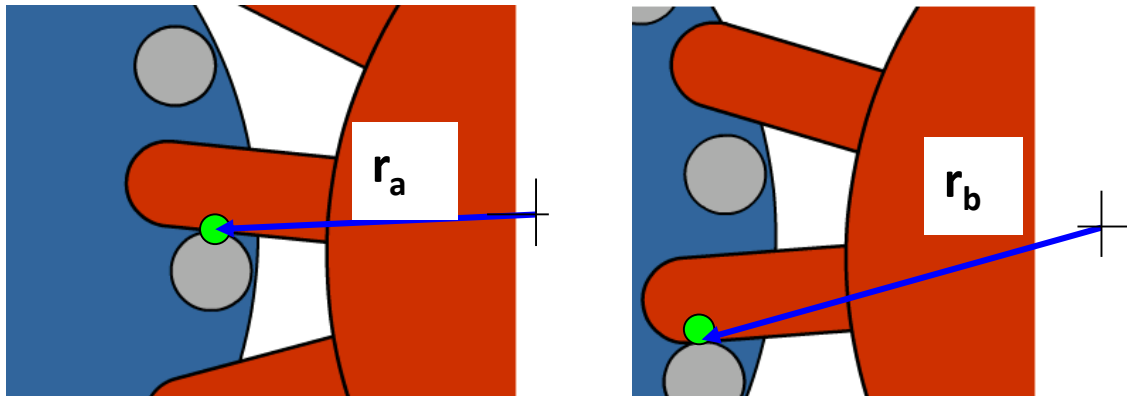
## Part 2



# Gear working mechanism: Conjugate action

**Definition:** When tooth profiles are designed to produce a constant angular-velocity ratio during meshing are said to have conjugate action.

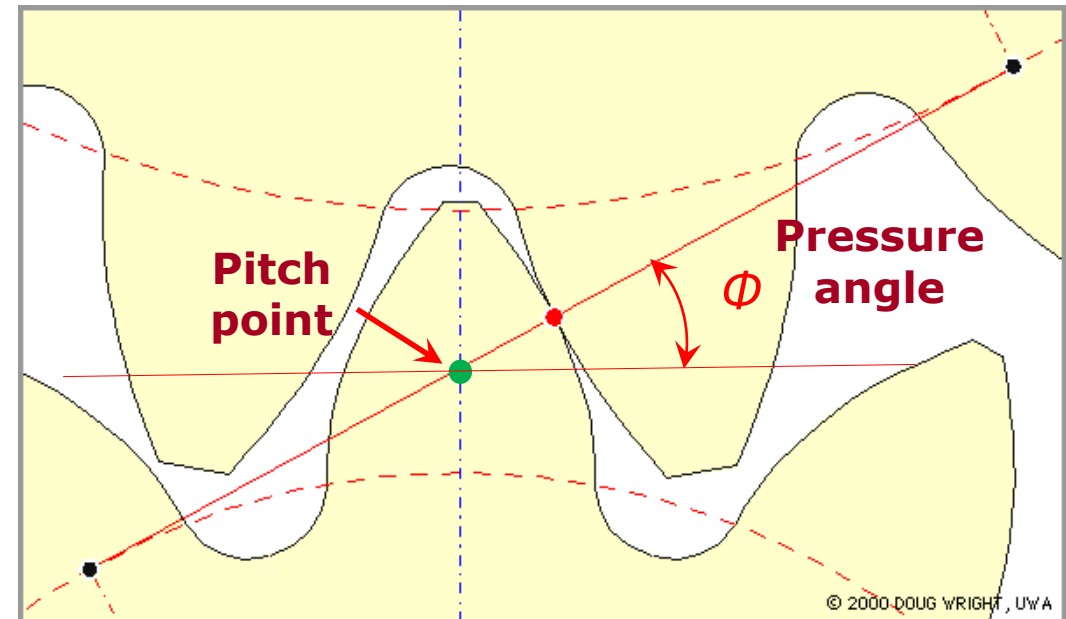
- The **common normal** to the surfaces at the point of contact always intersects the line of centres at the same Pitch Point, P.
- The point of contact moves along the **line of action**, tangent to the **base circles**.
- The **line of action** is always normal to the tooth profile at the point of contact.



$$r_b > r_a$$

Non-conjugate action of “Primitive” gears

[https://www.youtube.com/watch?v=xE\\_I3TE-2AY](https://www.youtube.com/watch?v=xE_I3TE-2AY)



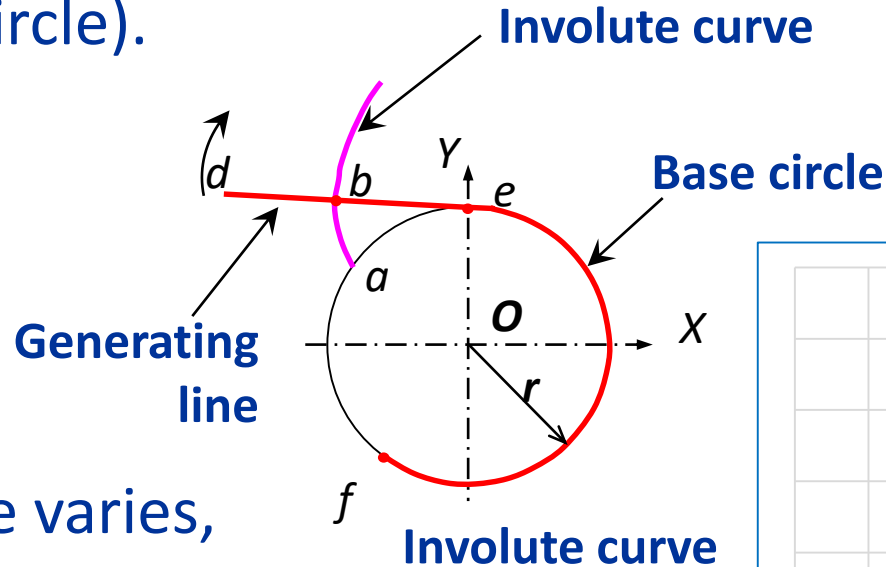
<http://www.mech.uwa.edu.au/DANotes/gears/meshing/meshing.html>

# Gear tooth profile: Involute curve

**Involute** is the curve traced by a point on a tight cord as the cord is unwound from a disc or circle (base circle).

$$x = r(\cos t + t \sin t)$$

$$y = r(\sin t - t \cos t)$$

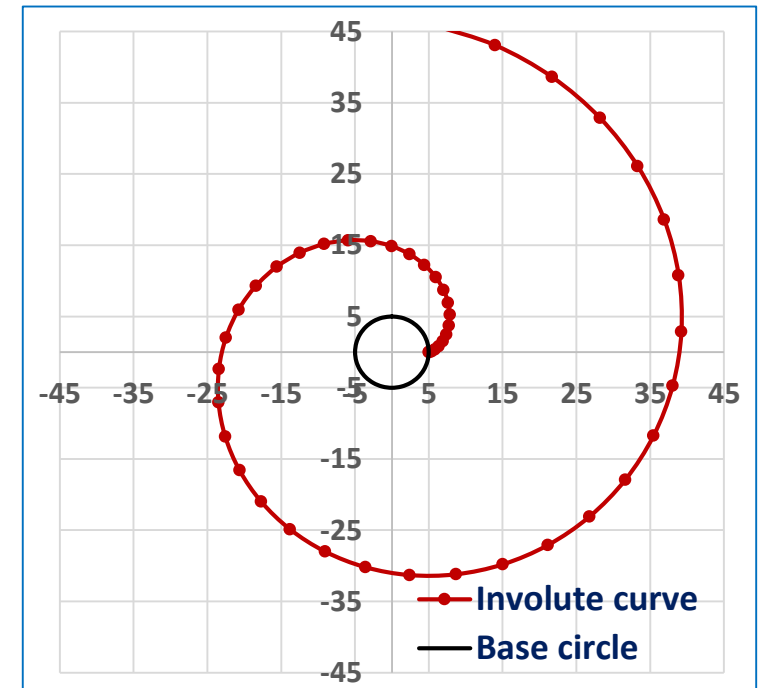


## Involute properties

- Radius of involute curvature varies,
- Involute doesn't exist inside the base circle,
- **Generating line** always **normal** to the involute & tangent to the base circle,



**Constant Speed Ratio**



[http://upload.wikimedia.org/wikipedia/commons/8/88/Animated\\_involute\\_of\\_circle.gif](http://upload.wikimedia.org/wikipedia/commons/8/88/Animated_involute_of_circle.gif)

# Gear ratio equation

$$V = \omega_p r_p = \omega_g r_g$$

$$Z = \frac{\omega_p}{\omega_g} = \frac{r_g}{r_p} = \frac{d_g}{d_p}$$

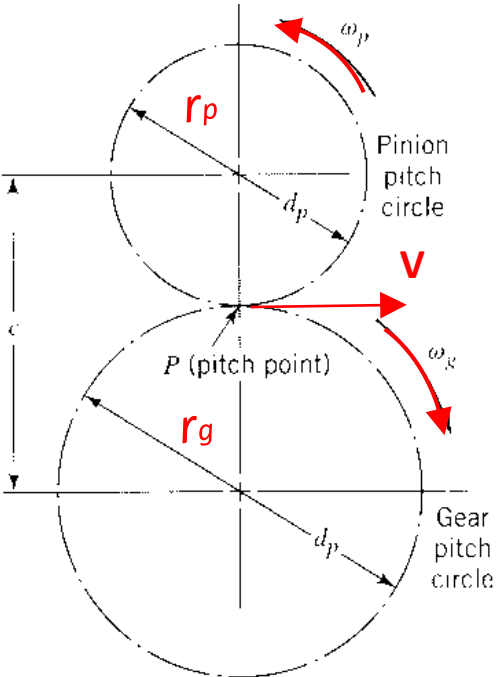
$$\left. \begin{aligned} V_{np} &= V_{ng} \\ V_{np} &= \omega_p r_{bp} \\ V_{ng} &= \omega_g r_{bg} \\ r_b &= r \cos \phi \end{aligned} \right\}$$



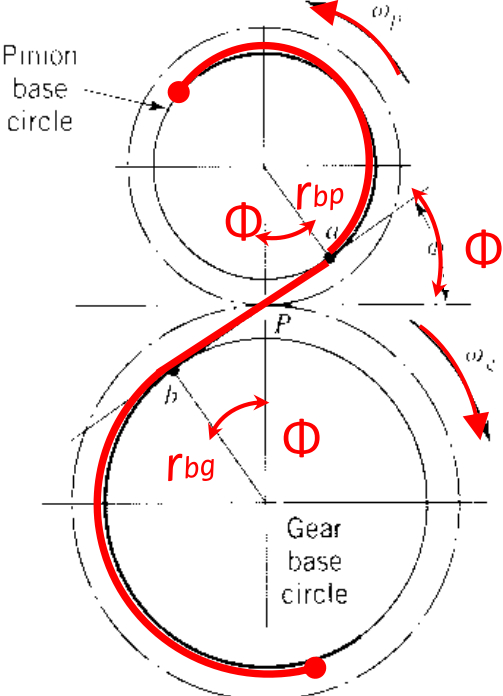
$$Z = \frac{\omega_p}{\omega_g} = \frac{r_{bg}}{r_{bp}} = \frac{r_g \cos \phi}{r_p \cos \phi} = \frac{d_g}{d_p}$$



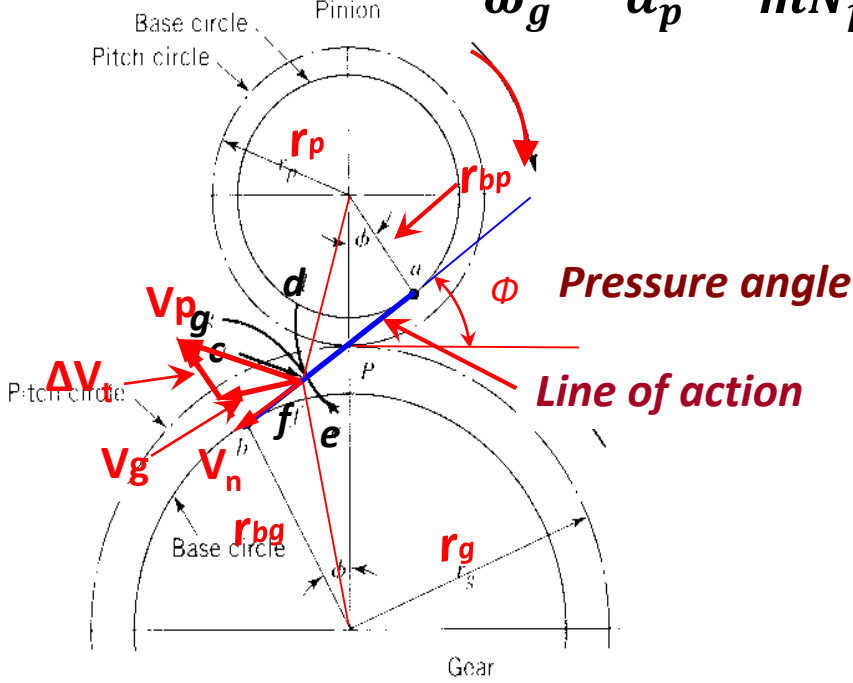
$$Z = \frac{\omega_p}{\omega_g} = \frac{d_g}{d_p} = \frac{m N_g}{m N_p} = \frac{N_g}{N_p}$$



Friction drive



Belt+friction drive



Involute gear drive

# Fundamental equations

Pitch diameter: (SI unit)  $d = mN$

Gear ratio (Z):  $Z = \frac{\omega_1}{\omega_2} = \frac{d_2}{d_1} = \frac{N_2}{N_1}$  Pressure angle,  $\phi$ , is another fundamental parameter

The necessary and sufficient conditions of proper meshing:

$$m_1 = m_2, \quad \phi_1 = \phi_2$$

*where, subscripts 1 & 2 are used to denote Pinion & Gear, respectively.*

Note:  $\phi = 20^\circ$  is the most commonly used pressure angle ( $\phi = 14.5^\circ$  is another option)

Important parameters for a pair of spur gears:  $m, \phi$  and  $N_1, N_2$

Centre distance (C):  $C = \frac{m}{2}(N_1 + N_2)$

Minimum number of teeth for standard gears with  $\phi = 20^\circ$  is 18.

# Gear geometry equations

<b>Formulas of gear geometry</b>			
Circular pitch, $p$	$p = \pi m$ (mm)	Working depth, $h_k$	$h_k = a + b - c = 2.0m$
Pitch diameter, $D$	$D = mN$ (mm)	Base circle diameter, $D_b$	$D_b = D \cos \phi$
Addendum, $a$	$a = 1m$ <sup>a)</sup>	Outside diameter, $D_o$	$D_o = D + 2a$
Dedendum, $b$	$b = 1.25m$ <sup>a)</sup>	Root diameter, $D_r$	$D_r = D - 2b$
Clearance, $c$	$c = b - a = 0.25m$ <sup>a)</sup>	Central distance, $C$	$C = \frac{m}{2} (N_1 + N_2)$
Whole depth, $h_t$	$h_t = a + b = 2.25m$	Tooth Thickness, $t$	$t = \frac{\pi}{2} m$
a) Referring to standard and commonly used tooth systems for spur gears			



# A simple example

For a pair of spur gears with the module of  $m=3 \text{ mm}$ , the numbers of teeth for the pinion and the gear are  $N_1=20$  and  $N_2=35$ , respectively. The pressure angle is  $20^\circ$ . Determine the parameters of both the **pinion and gear**.

Calculation of pinion parameters ( <i>unit in mm</i> )			
Pitch diameter, $D$	$D=mN_1=3 \times 20=60$	Working depth, $h_k$	$h_k=a+b-c=2.0m=2 \times 3=6$
Circular pitch, $p$	$P=\pi m=3.1416 \times 3=9.425$	Base circle diameter, $D_b$	$D_b=D \cos \phi=60 \times \cos 20^\circ=56.382$
Addendum, $a$	$a=1m=1 \times 3=3$	Outside diameter, $D_o$	$D_o=D+2a=60+2 \times 3=66$
Dedendum, $b$	$b=1.25m=1.25 \times 3=3.75$	Root diameter, $D_r$	$D_r=D-2b=60-2 \times 3.75=52.5$
Clearance, $c$	$c=b-a=0.25m=0.25 \times 3=0.75$	Central distance, $C$	$C=1/2m(N_1+N_2)=0.5 \times 3(20+35)=82.5$
Whole depth, $h_t$	$h_t=a+b=2.25m=2.25 \times 3=6.75$	Tool thickness, $t$	$t=1/2\pi m=0.5 \times 3.1416 \times 3=4.7124$

# Gear Tooth Systems

A **tooth system** is a standard to specify gear tooth geometry. The standard is used to attain interchangeability of gears of all tooth numbers of the same **Pressure Angle ( $\phi$ )** and **Module ( $m$ )**, as shown in the Table.

**Key parameters of spur gears are**

**$m, N, \phi$  (either  $20^\circ$  or  $14.5^\circ$ )**

**•Tooth sizes in General Uses (SI system) (BS ISO 54:1996)**

Module, ( $m$ ) – SI unit (mm)	
Preferred	<b>0.5, 0.8, 1, 1.25, 1.5, 2, 2.5, 3, 4, 5, 6, 8, 10, 12, 16, 20, 25, 32, 40, 50</b>
Next choice	<b>1.125, 1.375, 1.75, 2.25, 2.75, 3.5, 4.5, 5.5, 7, 9, 11, 14, 18, 36, 45</b>

# Summary

- Understand the main function and different types of gears as well as how gear systems are classified;
- Be familiar with gear terminologies and be able to determine gear ratio and key parameters with a given simple gear train (system);
- Understand the concepts of Involute profile, conjugate action, gear tooth system and fundamental requirement for proper gear meshing.

# Revision questions

- How are different types of gears **classified**?
- Can you name a few specific forms of gears between two parallel shafts?
- How do you know the **pitch diameter ( $d$ )** of a gear with given **Module ( $m$ )** and **Number of teeth ( $N$ )**?
- What are the key characteristics of **conjugate action** in gear meshing?
- What are the **key parameters** in gear design?
- What are the **necessary and sufficient conditions** for a pair of gears meshing properly?
- Why is it useful to have **Gear tooth system** in practice?
- What is the **gear ratio equation** of a pair of gears?

# Gears references and resources

- R.G. Budynas and J.K. Nisbett, Shigley's Mechanical Engineering Design, 10th ed., 2015, (TJ230 BUD)
- Childs, P.R.N., Mechanical Design Engineering Handbook, 2nd ed., 2019, (TJ230 CHI)
- <http://en.wikipedia.org/wiki/Gear>
- <http://www.geartechnology.com>
- <http://powertransmission.com>





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# Gears 1

## End of Part 2



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# Gears 1

## Part 3

# Worked examples of Gears 1

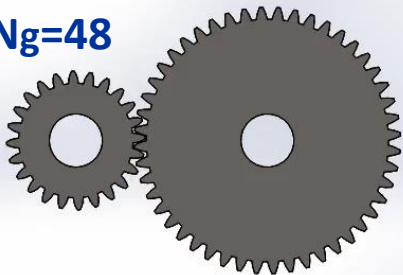
## Part 3:

- **Evaluation of simple gear train and gear parameters** as included in Gears 1 handouts available on Moodle
- **Use of SW Design Library/Toolbox and external download from GrabCAD, (<https://grabcad.com/>)** to build gear assembly models and animations

# Worked example 2: Using **SW Design Library/Toolbox** to build Gear Assembly models

- **Solidworks Design Library/Toolbox** allows access of various types of gears
  - **SW Mechanical Mate function** allows gear ratio based motion relations to be defined.
  - **SW Motion Analysis** module allows simple animation and detailed motion calculations to be fully explored.

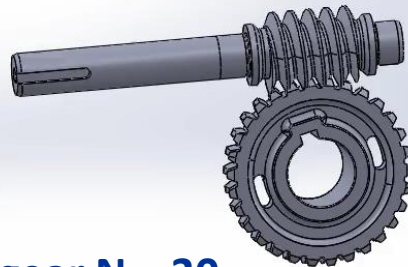
Spur gears  
 $N_p=24, N_g=48$



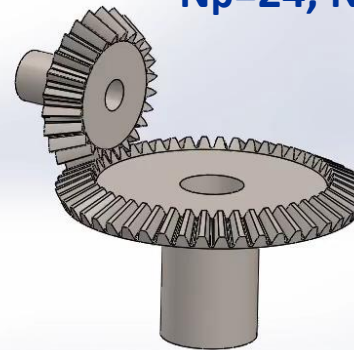
Worm & gear  $N_g=30$

Downloaded from GrabCAD

<https://grabcad.com/>



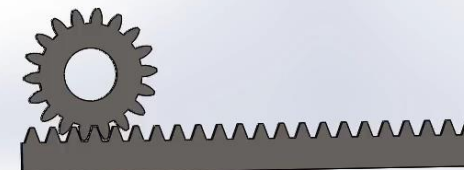
Bevel gears  
 $N_p=24, N_g=48$



Internal spur gears  
 $N_p=18, N_g=72$



Rack & pinion  
 $N_p=20$



**Note:** SW assembly models of gears are available on Moodle [SW motion analysis of gears youtube tutorial](https://www.youtube.com/watch?v=gYJVr4QuzAE)  
<https://www.youtube.com/watch?v=gYJVr4QuzAE>



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# Gears 1

## End of Session